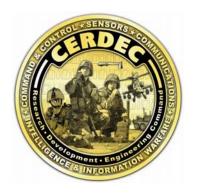
Power Generation and Alternative Energy Branch

US Army RDECOM CERDEC CP&ID Power Division Aberdeen Proving Ground, MD





PGAE - TR - 11 - 15

Hybrid Vapor Compression Ejector Cycle: Presentation to IAPG Mechanical Working Group

Parmesh Verma and Tom Radcliff, United Technologies Research Center

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Hybrid Vapor Compression Ejector Cycle

Final Review Parmesh Verma and Tom Radcliff Aug. 2011



United Technologies Research Center – East Hartford, CT



Outline

CO₂ Hybrid Vapor Compression Ejector Cycle

System Overview

Component and System Design

Integrated Controls

Fabrication and Testing: Advance Technology Demonstrator (ATD)

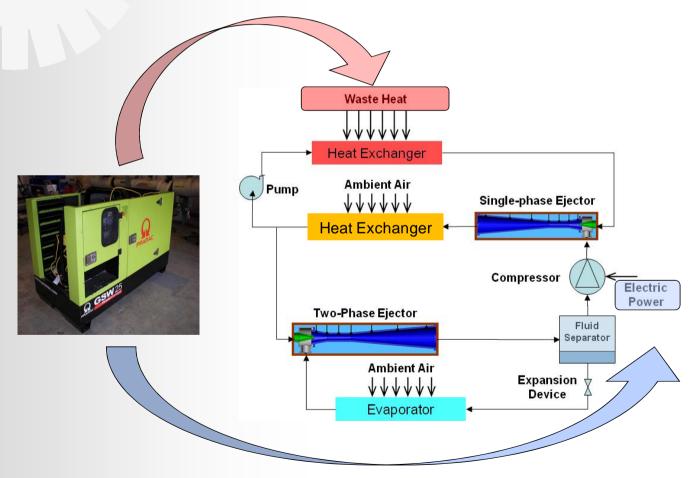
Plan



System Overview

Objective: Design, develop & demonstrate a CO₂ 5TR (@ 125F ambient) environmental control unit (ECU) using ejector top & bottom cycles

Deliverables: +10% η_{system} vs. R410A (baseline); ATD; and reduced-order design tools





Development Process

Model based process used to optimize system configuration and component design





ATD and Models



Testing and Validation



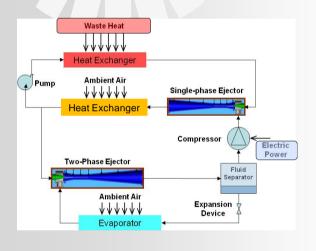
✓ATD fabrication and commissioning



System and Component Design

Steady-state system model developed in EES and used to design components

System Modeling and Analysis

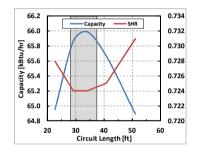


System modeled in EES

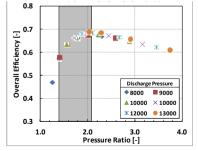


System Schematic

Component Design



Heat Exchanger



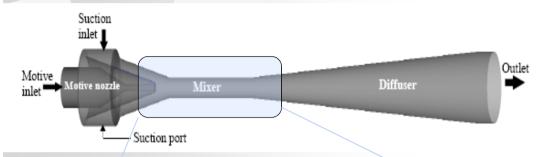
Compressor

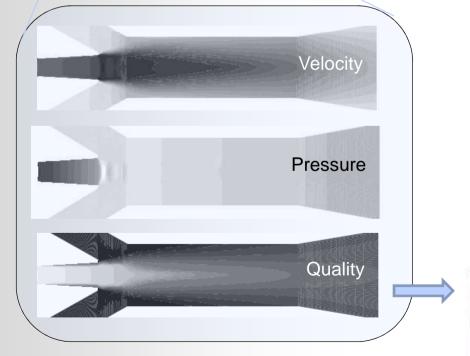


Work Recovery Ejector Design

Physics based CFD model developed and validated against multiple experimental

data used to design work recovery ejector





United Technologies
Research Center

Bubble growth: inertial and thermal

$$\dot{m}_{\rm cav,e} = C_e \frac{\sqrt{k}}{\sigma} \rho_l \rho_v \left[\frac{2}{3} \frac{P_v - P}{\rho_l} \right]^{1/2} (1 - \chi_v - \chi_g)$$

$$\dot{m}_{\text{boil}} = \pm \left[\frac{\hat{\sigma}}{2 - \hat{\sigma}} \right] \left(\frac{M}{2\pi R T_{\text{sat}}} \right)^{1/2} [P - P_v]$$

Drift-flux phase slip

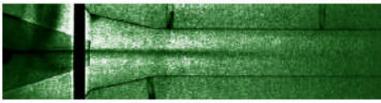
$$\mathbf{v}_{dr,p} = \mathbf{v}_{pq} - \frac{1}{\rho} \alpha_k (\rho \mathbf{v}_q)_k$$

2-phase sonic speed

$$\frac{1}{\rho c^2} = \frac{\alpha_v}{P} \left[(1 - \epsilon_v) f_v + \epsilon_v g_v \right] + \frac{1 - \alpha_v}{P} \epsilon_l g_l$$

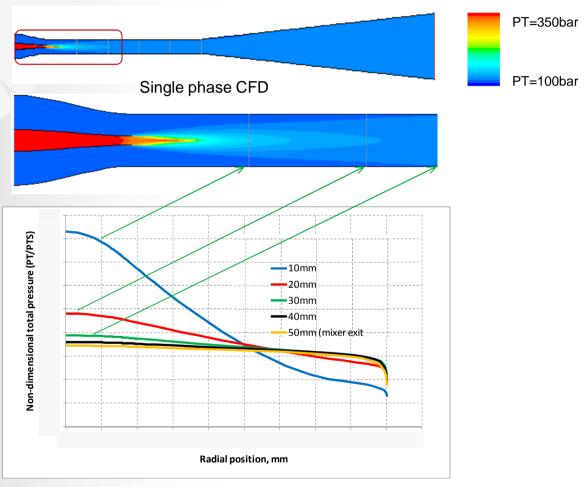
Model equations as published in International Journal of Heat and Mass Transfer (v.55, 2012)

Neutron image validation



Heat Recovery Ejector Design

Mixing limits design but simulation can be used to investigate new alternatives

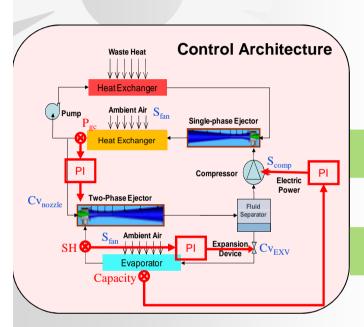


Motive momentum dissipates through turbulence too early in the mixing process when entrainment ratio is very high

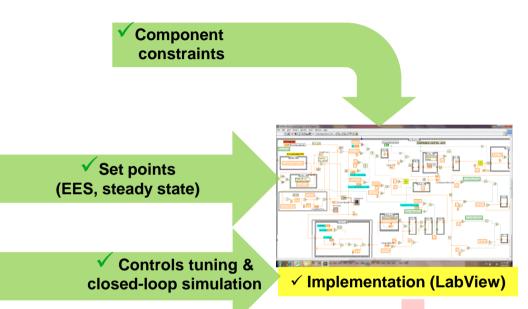


Integrated Controls

Automatic synchronization of three loops to maximize system efficiency



Actuator	Controlled variable
Compressor	Capacity
Ejector	Gas cooler pressure
EXV	Superheat
Gas cooler fan	Indirectly for COP

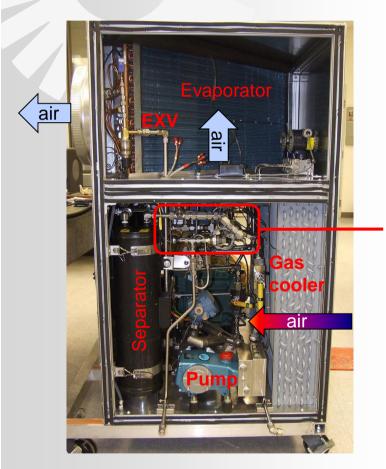




✓ Control Performance Verification

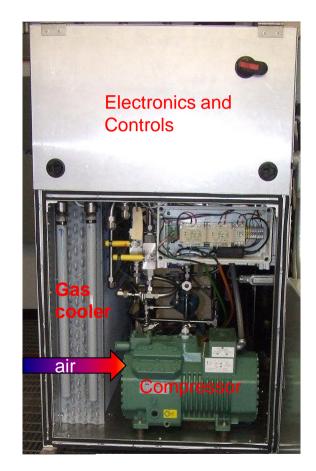
Advance Technology Demonstrator (ATD)

Advance technology demonstrator fabricated



Heat and work recovery ejectors

Back

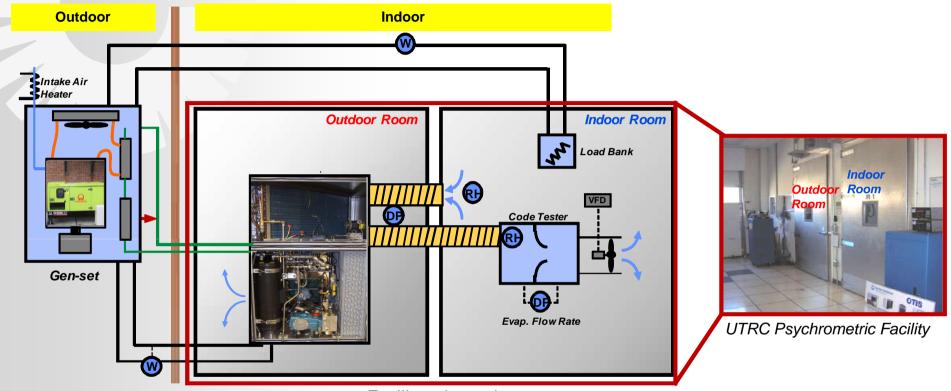


Front



Advance Technology Demonstrator (ATD)

ATD commissioned at Psychrometric Facility



Facility schematic

	Test Conditions for Indoor and Outdoor rooms				
	Test Conditions	Indoo	Outdoor Room		
		Dry bulb (°F)	Dew Point (°F)	Dry bulb (°F)	
	ARI B	80	60.3	82	
	ARIA	80	60.3	95	
United Technologies Research Center	Army 125	90	75	125	
nesearch center					

Tested 3 modes:

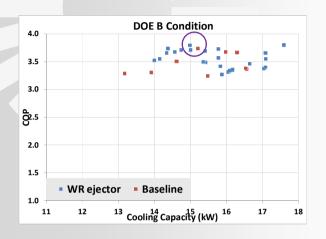
simple cycle, work recovery, work & heat recovery

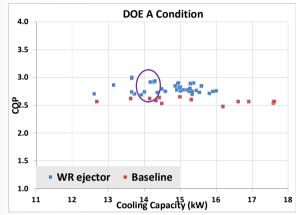
Other tests include:

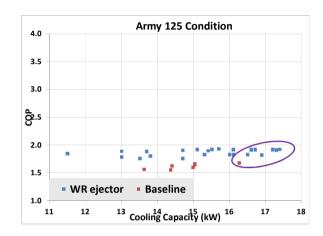
Low ambient and endurance/controls reliability

Test Results Summary

Up to 16% improvement in system efficiency demonstrated with work recovery ejector cycle







Condition		Cooling	System Efficiency (COP)	
		Cooling (kW)	Testing (Measured)	
DOE B (82°F)	WR & HR	14.7	3.21	
	WR ejector	15.0	3.80	2% ↑
	Baseline	15.2	3.74	-
DOE A (95°F)	WR & HR	15.5	2.61*	
	WR ejector	14.3	2.93	11% 🕆
	Baseline	14.4	2.65	-
Army (125°F)	WR ejector	17.4	1.93	16% ↑
	Baseline	15.0	1.66	-



Performance Metrics

Performance: Demonstrated 2x COP improvement vs. current ECU but 30% short of target

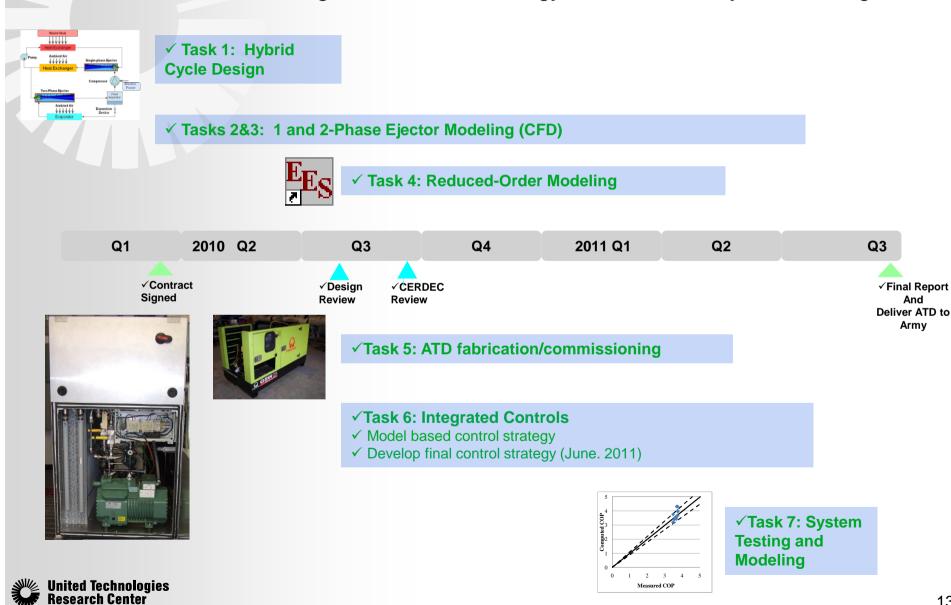
Size: Current ATD comparable to existing ECU but exceeds targets. Optimized ATD weight within target

Name	Typical Current ECU	Contract Target	Current ATD	Improved ATD w/o Heat Recovery
System efficiency @ 125F	~1	2.75	1.93 (measured)	similar to current ATD
Size	39"x28"x72" 45.5 cu ft	36"x36"x48" 36 cu ft	32"x43"x57" 45.4 cu ft	32"x43"x50" 40 cu ft
Weight	<750 lbm	target 600 lbm (stretch 400 lbm)	934 lbm	< 500 lbm



Contract Plan

Demonstrated TRL 5 through advance technology demonstrator system testing



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